



University of Liege  
Department ArGENCO



# De Paepe – Willems Award

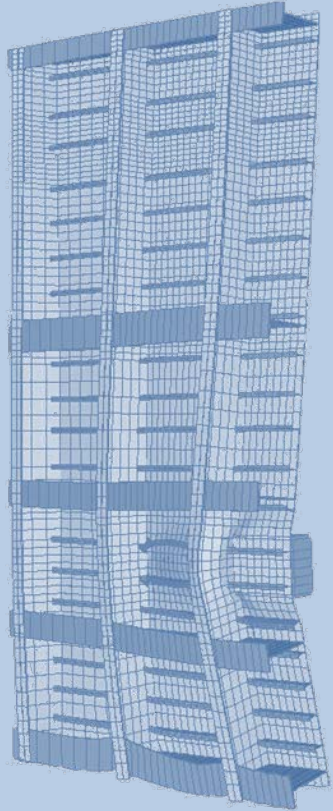
*Optimization and analysis of lock gates in the framework  
of the “Seine-Scheldt-East” waterway upgrading*

THOMAS GERNAY

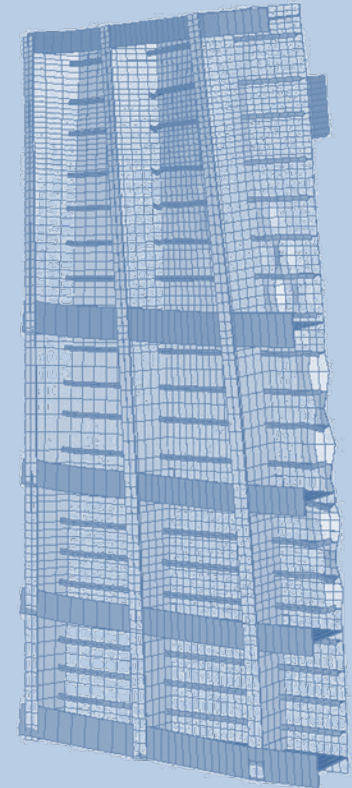
Research Fellow F.R.S.-FNRS  
University of Liege, Belgium

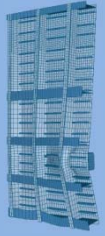
PIANC MMX AGA – 10 May 2010 – Liverpool





# PLAN

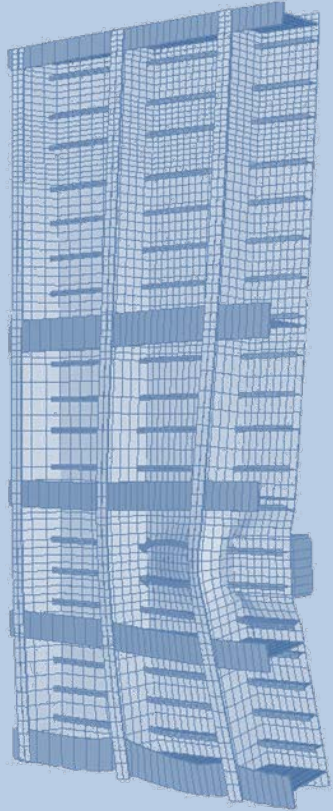




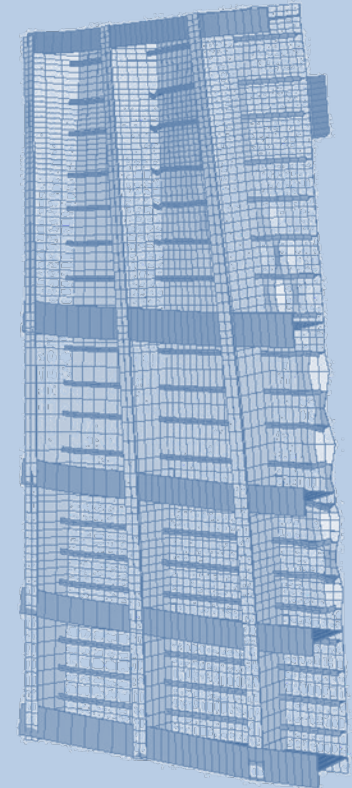
1. Introduction
2. Design & Optimization
3. FEM
4. Ship impact
5. Ship impact results
6. Conclusion

## Plan

- 1. Introduction**
- 2. Design & Optimization of the lock gate**
- 3. Finite Element Model (FEM)**
- 4. Ship impact**
- 5. Ship impact results**
- 6. Conclusion**



# Introduction







1. Introduction
2. Design & Optimization
3. FEM
4. Ship impact
5. Ship impact results
6. Conclusion

# Introduction

“Seine-Scheldt-East” project

“Seine-Nord Europe” canal connecting the Seine river to the European waterway network

- ✓ Northern Europe
- ✓ Rhin-Main-Danube to Black Sea
- ✓ Le Havre, Rouen with Dunkerque, Anvers, Rotterdam



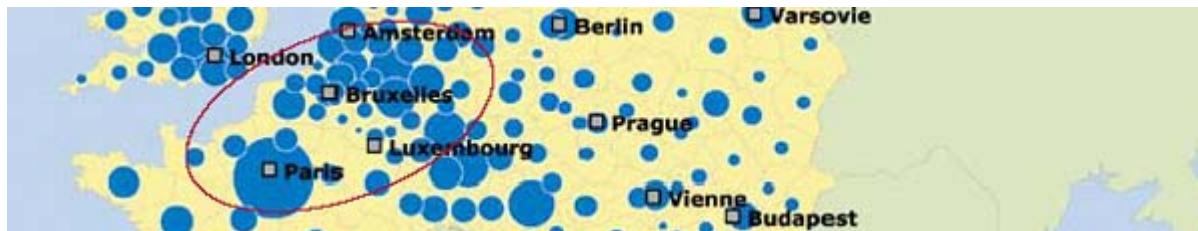


1. Introduction
2. Design & Optimization
3. FEM
4. Ship impact
5. Ship impact results
6. Conclusion

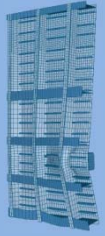
## “Seine-Scheldt-East” project



Important zone in terms of population (EUROSTAT 2003)



Important zone in terms of GDP (EUROSTAT 2003)



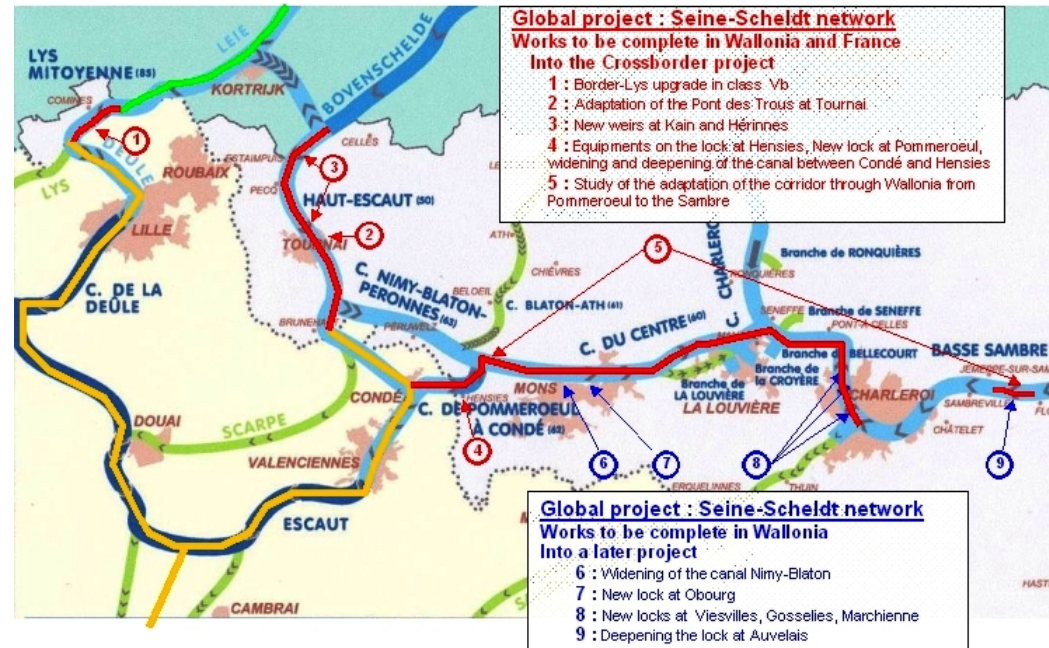
1. Introduction
2. Design & Optimization
3. FEM

4. Ship impact
5. Ship impact results
6. Conclusion

## “Seine-Scheldt-East” project

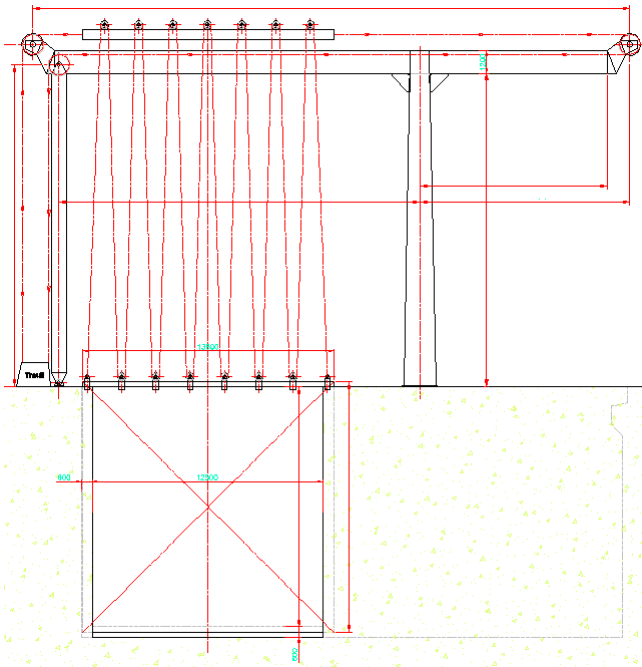
### 4 new locks planned by the Walloon Region

- ✓ Connecting the Scheldt and the Meuse rivers
- ✓ Marchienne-au-Pont, Gosselies, Viesville and Obourg
- ✓ Class Va in Europe (2000 tons)
- ✓ Lock 112,5 m x 12,5 m





## Analysis of the downstream gates of the four new locks



### Decisions from the basic preliminary design:

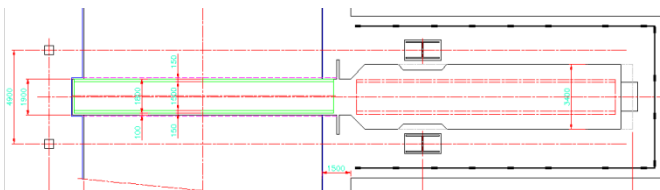
- ✓ Gates suspended and manoeuvred by lateral movement
- ✓ Standardization → focus on one gate

### Dimensions

- Length: 13,70 m
- Height: 13,60 m
- Width:  $1,00 \text{ m} < l < 1,80 \text{ m}$

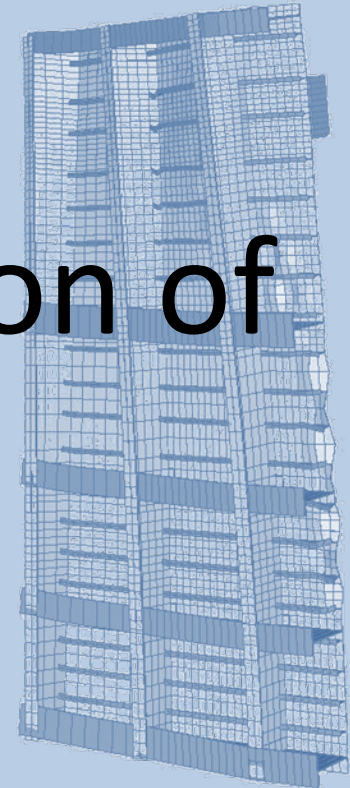
### To be determined

- ✓ Width
- ✓ Use of waterproof compartment
- ✓ Gate structure (elements, thicknesses, ...)





# Design & Optimization of the lock gate







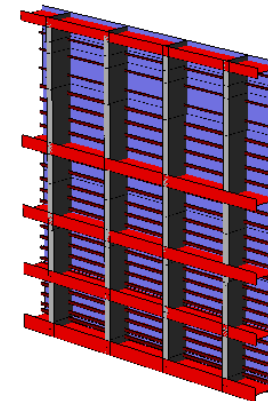
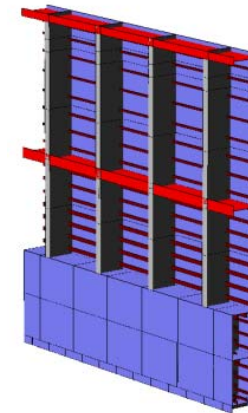
1. Introduction
- 2. Design & Optimization**
3. FEM

4. Ship impact
5. Ship impact results
6. Conclusion

# Design & Optimization

## Analysis of 4 different models

- 2 models with additional waterproof compartment and different gate widths
- 2 models without waterproof compartment, with different gate widths





1. Introduction
2. Design & Optimization
3. FEM

4. Ship impact
5. Ship impact results
6. Conclusion

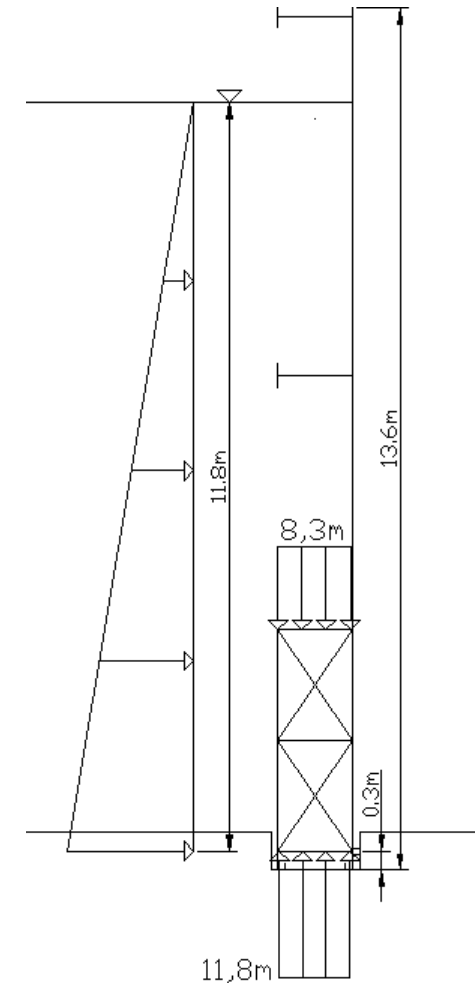
## Design & Optimization

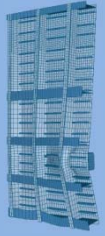
Optimization (weight, cost) of the structure considering hydrostatic load cases

→ Elastic design  $\sigma < 176 \text{ Mpa}$

Slenderness ratio of reinforcement elements (frames, girders, stiffeners) adapted to avoid instabilities (Huges criteria)

Exceptional hydrostatic load case: downstream section of the canal empty

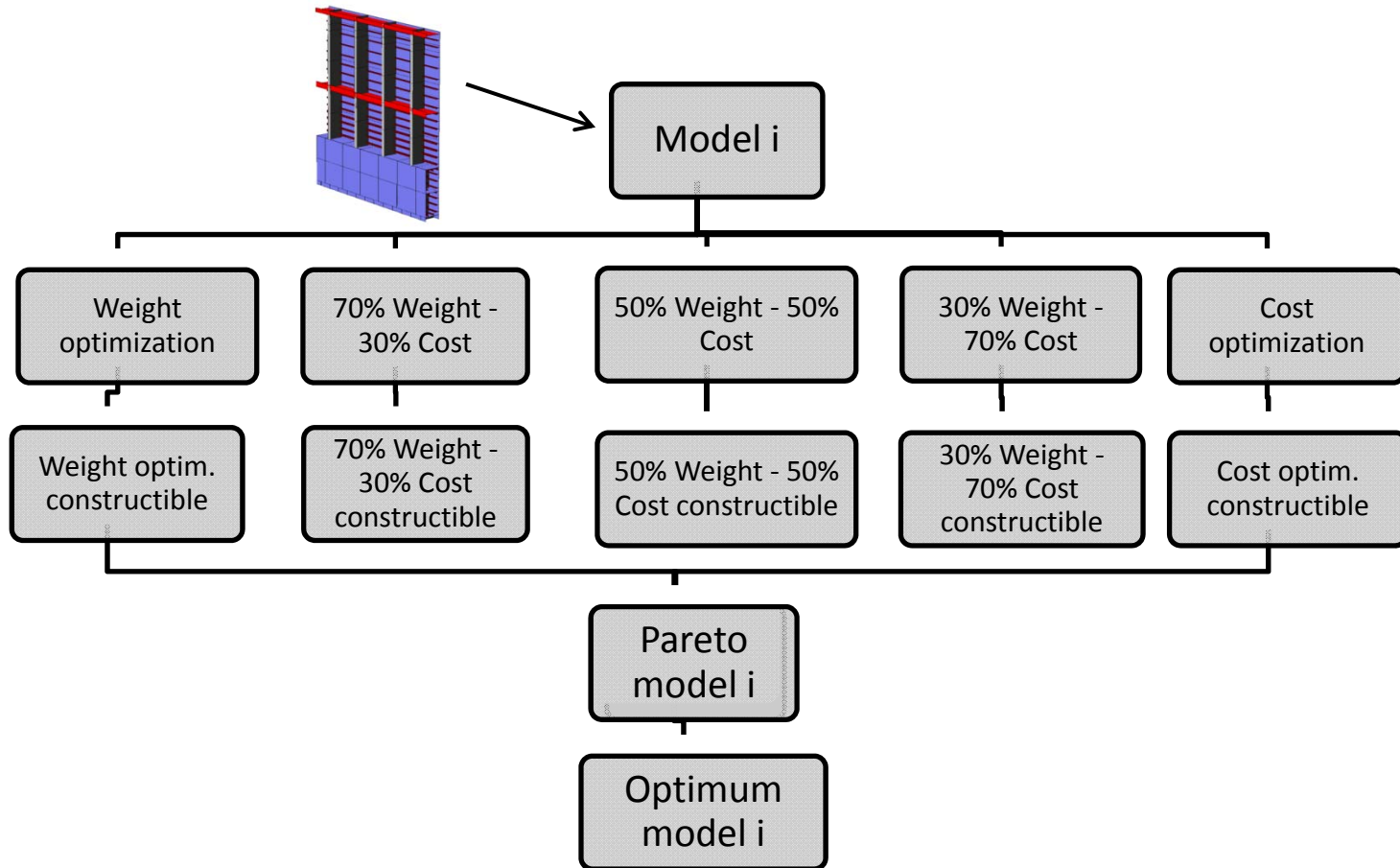




1. Introduction
2. Design & Optimization
3. FEM

4. Ship impact
5. Ship impact results
6. Conclusion

## Optimization process in successive stages using the LBR5 software (Ph. Rigo, Ulg)



=> Optimum solution for one model

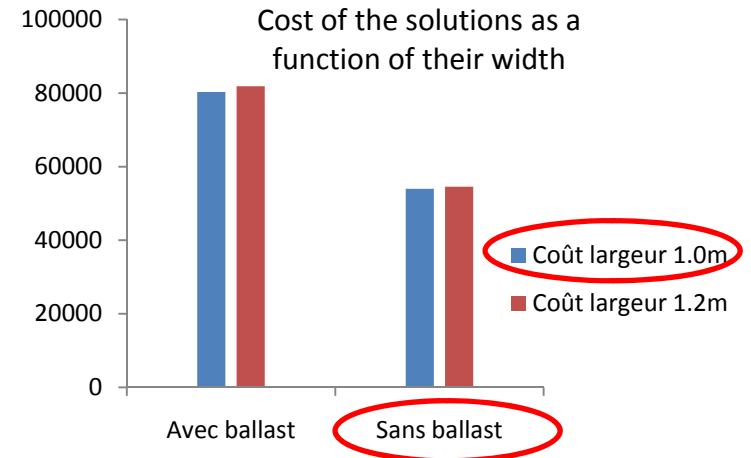
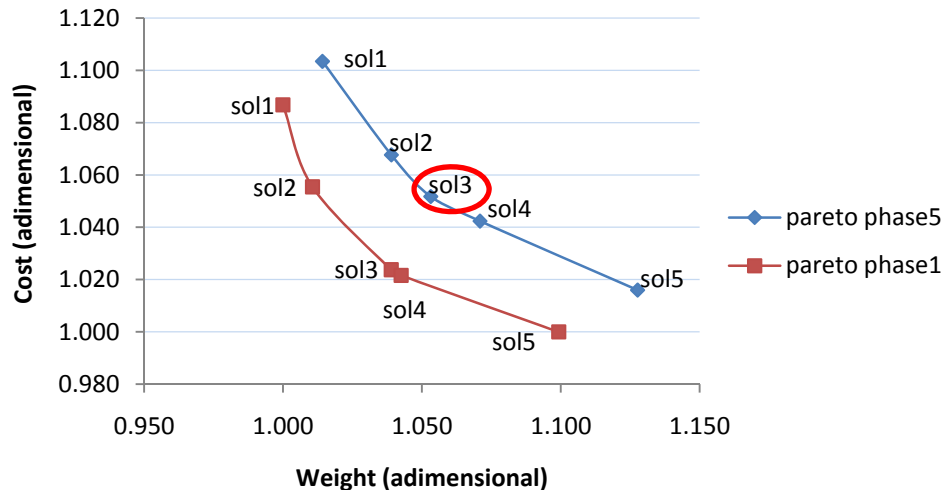


1. Introduction
2. Design & Optimization
3. FEM

4. Ship impact
5. Ship impact results
6. Conclusion

## Pareto curve for model i

- ✓ Considers weight and cost
- ✓ Define a criteria of selection to pick the optimum solution



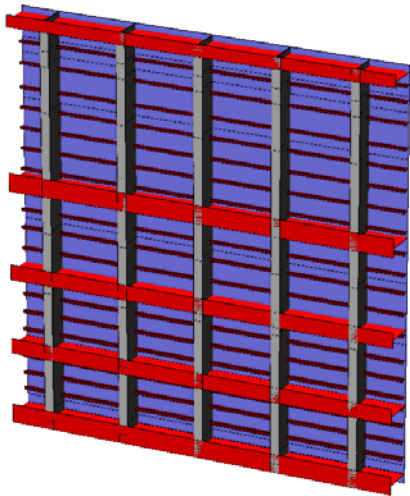
## Comparison between the optimum solutions of the 4 models

- ✓ Choice of design: width 1.0 m without waterproof compartment

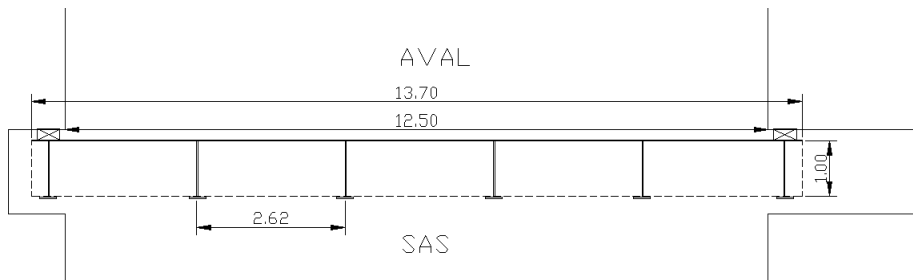


1. Introduction
2. Design & Optimization
3. FEM
4. Ship impact
5. Ship impact results
6. Conclusion

## Final design solution: analysis for all hydrostatic cases



LBR5 Analysis	In service conditions		In flood conditions		Downstream side empty	
	1	2	1	2	1	2
Orientation of the gate						
Girders	151 MPa		168 MPa		159 MPa	
Stiffeners	156 MPa		168 MPa		171 MPa	
Frames	157 MPa		169 MPa		174 MPa	
Plate (thick./min. th. ratio)	1.16	1.08	1.11	1.04	1.13	1.06
Deflection	1.83 cm		2.04 cm		1.94 cm	

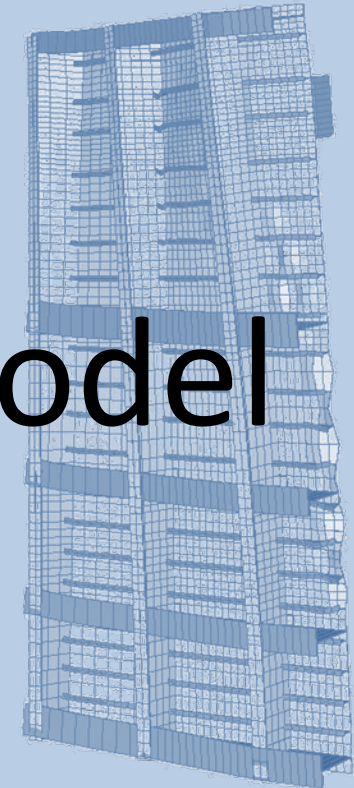


Solution	
Weight	51.4 t
Cost	56 202 €





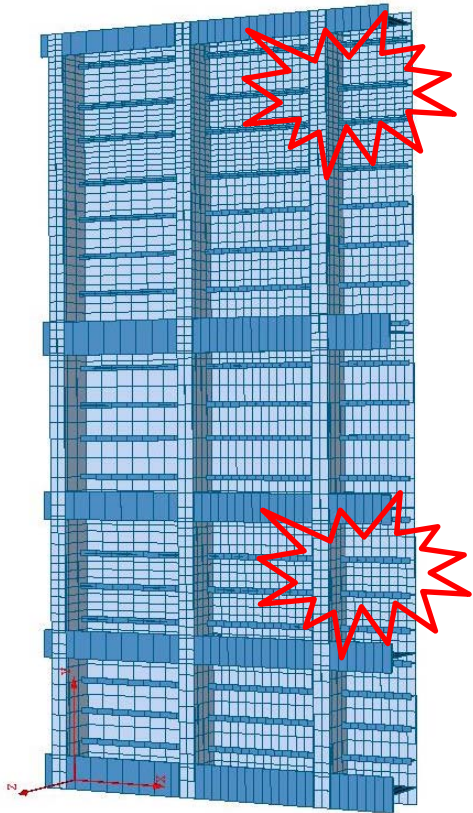
# Finite Elements Model



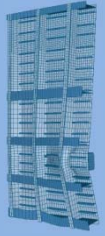


1. Introduction
2. Design & Optimization
3. **FEM**
4. Ship impact
5. Ship impact results
6. Conclusion

## FEM of the solution



- ✓ Half of the gate
- ✓ FINELG software (V. de Ville, Ulg)
- ✓ Conform to the LBR5 model
- ✓ Simply supported on three sides
- ✓ Refined mesh in the impact zones



1. Introduction
2. Design & Optimization
3. **FEM**
4. Ship impact
5. Ship impact results
6. Conclusion

## Linear elastic analysis for hydrostatic loading

Comparison between LBR5 and FINELG results

- ✓ Good concordance except for the maximal deflection
- ✓ Validate the finite elements model

Loading	C1 (service)		C2 (flood)		C3 (downstr. empty)	
Software	LBR-5	FINELG	LBR-5	FINELG	LBR-5	FINELG
Max. stress in frames	157 MPa	149 MPa	169 MPa	157 MPa	174 MPa	171 MPa
Maximal deflection	1.83 cm	2.19 cm	2.04 cm	2.44 cm	1.94 cm	2.31 cm

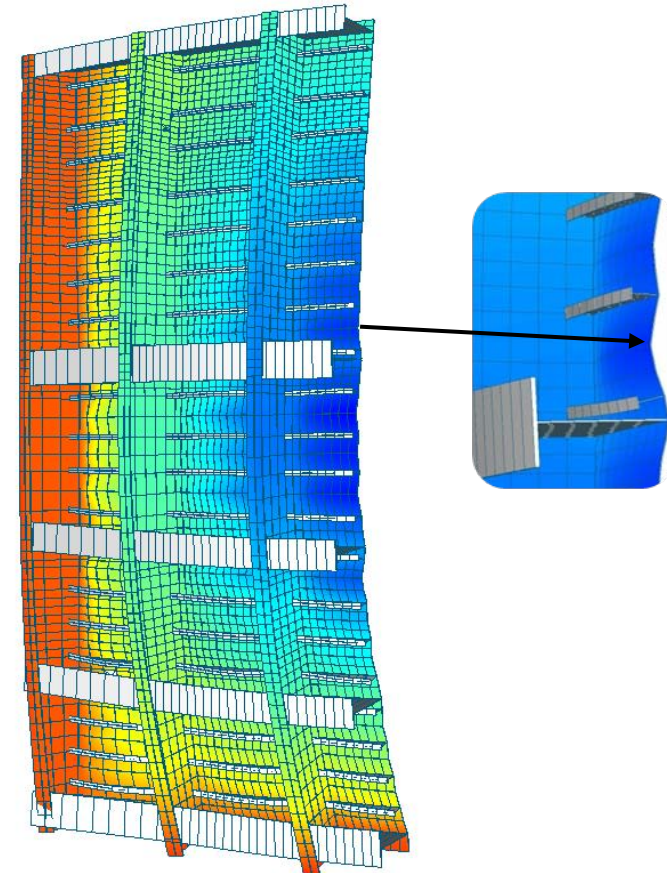
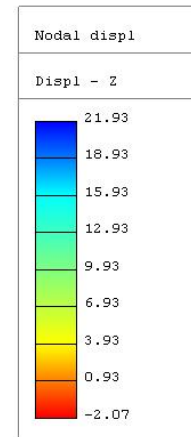


1. Introduction
2. Design & Optimization
3. FEM

4. Ship impact
5. Ship impact results
6. Conclusion

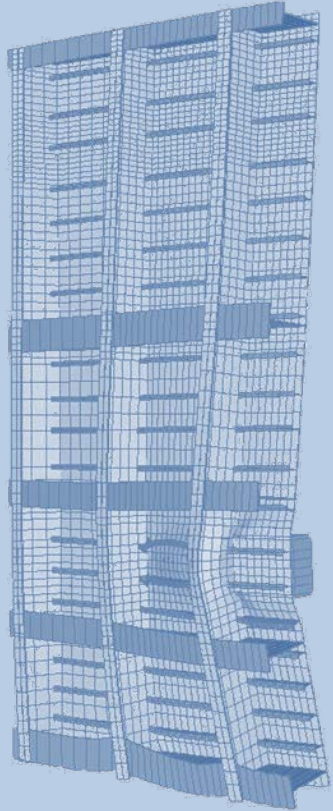
## Differences of maximal deflection?

- ✓ Local bending of the plate (not in LBR5)
- ✓ Same plate deflection along the girders with LBR5 and FINELG (no local bending)
- ✓ FINELG gives bigger plate deflection in the middle of unstiffened plate

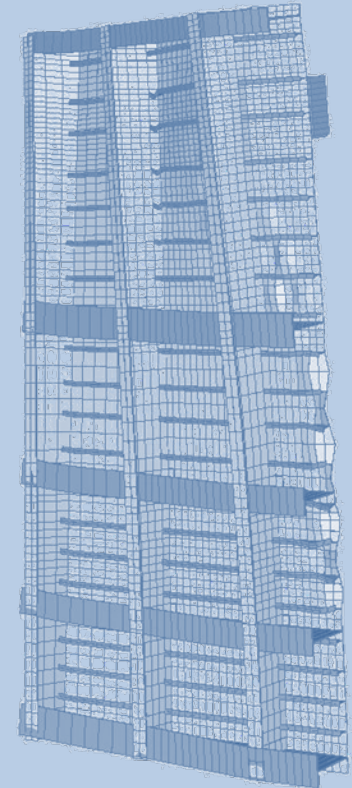


Deflection shape (mm) for hydrostatic service loading (amplified x80)





# Ship impact







1. Introduction
2. Design & Optimization
3. FEM

4. **Ship impact**
5. Ship impact results
6. Conclusion

## Ship impact analysis

### 1. Vessel impact design criterion?

Class 5a (2000 tons). Speed?  
No loss of watertightness and global resistance

### 2. Protective measures VS gate designed to sustain ship impact?

Compare additional cost of both solutions  
→ additional cost to provide the gate a sufficient impact strength?

### 3. In this analysis: Gate = ship stopping device

Structure must combine sufficient flexibility with sufficient load bearing capacity to successfully absorb the kinetic energy



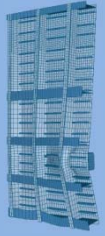
1. Introduction
2. Design & Optimization
3. FEM

4. **Ship impact**
5. Ship impact results
6. Conclusion

**Gate = ship stopping device**

**Analysis to perform to design the gate structure?**

- a) Empirical approach
- b) **Analytical-Rational approach**
- c) **FEM, quasi-static analysis**
- d) FEM, dynamic analysis



1. Introduction
2. Design & Optimization
3. FEM

4. **Ship impact**
5. Ship impact results
6. Conclusion

## FEM, quasi-static analysis

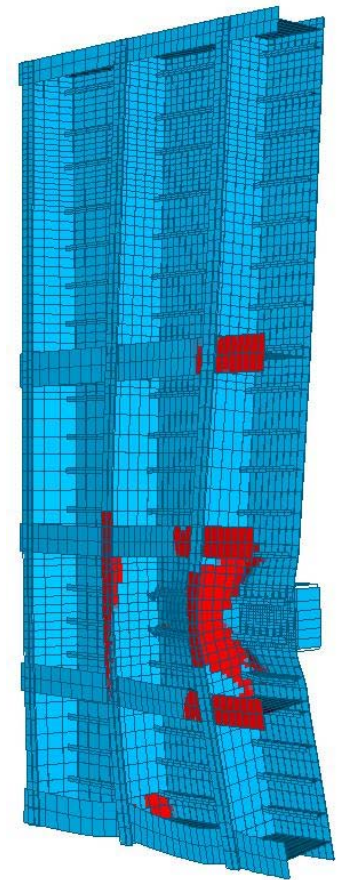
Finite Elements Method

Neglect the dynamic effects → quasi-static analysis

Hypothesis: infinitely stiff barge bow (in fact, the ship dissipates between 5 and 15 % of the initial kinetic energy [Le Sourne et al., 2003])

Simple model of the bow of the ship as a perfectly stiff rectangular element with no evolution of the contact between the bow and the gate

The load  $F_{impact}$  on the bow is increased until equalization of the energies



Lock gate simply supported on three sides

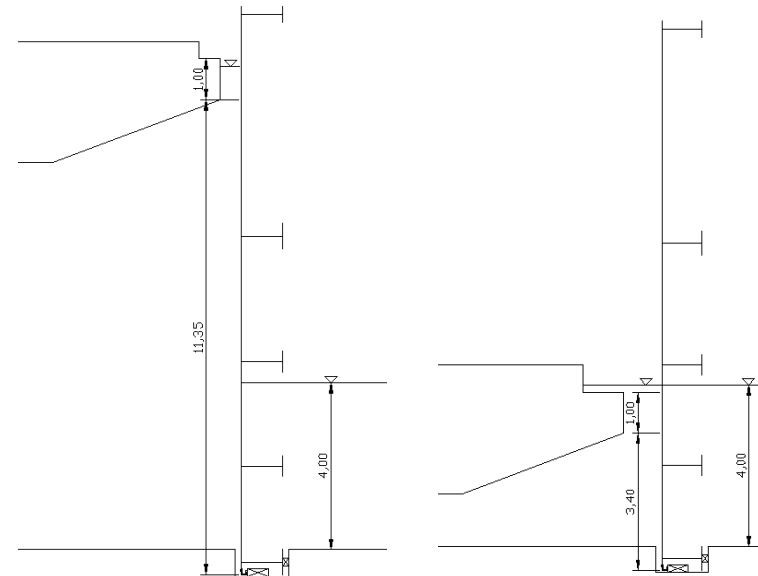


1. Introduction
2. Design & Optimization
3. FEM

4. Ship impact
5. Ship impact results
6. Conclusion

## Ship impact analysis

- Initial kinetic energy transformed into strain energy of the gate
- Elastic perfectly plastic material law for steel
- Analysis of 3 scenario
  1. Impact at upstream water level (U.W.L.), without hydrostatic load
  2. Impact at upstream water level, with hydrostatic load
  2. Impact at downstream water level (D.W.L.)





1. Introduction
2. Design & Optimization
3. FEM
4. Ship impact
5. Ship impact results
6. Conclusion

## 1st scenario: impact at U.W.L.

### Initially optimized structure

- Elastically designed with LBR5
- Hugues' geometric criteria for T elements (slenderness ratio to avoid instability)
- Hugues' criteria fit with **EC3**
  - Elastic bending moment OK but buckling before plastic bending moment
  - Low capacity of rotation
  - Inadequate to take advantage of the plastic field

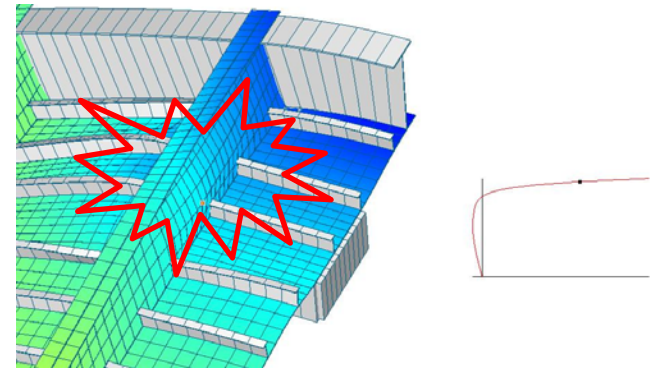
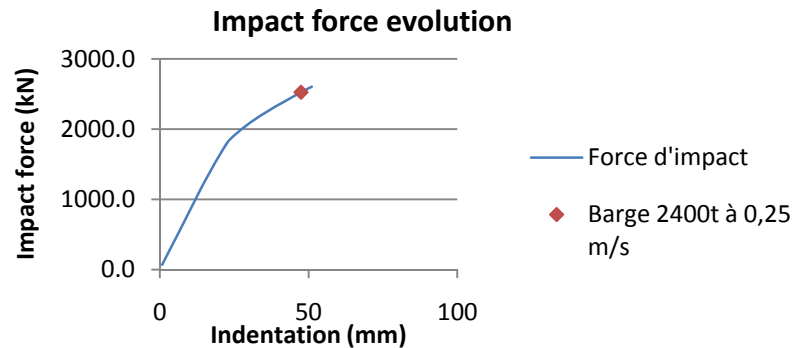
Modèle de comportement	Mom de résistance	Capacité de rotation	Classe
	Moment plastique sur section brute 		1
	Moment plastique sur section brute 		2
	Moment élastique sur section brute 		3
	Moment plastique sur section efficace 		4

$M_{el}$  moment de résistance élastique de la section transversale  
 $M_{pl}$  moment de résistance plastique de la section transversale  
 $M$  moment appliqué  
 $\phi$  rotation (courbure) de la section  
 $\phi_{pl}$  rotation (courbure) de la section exigée pour générer une distribution plastique totale des contraintes dans la section transversale



## 1st scenario: impact at U.W.L.

- With the initially optimized structure (class 3)



- ⇒ Instability (frame buckling)
- ⇒ Fragile global behaviour
- ⇒ Low capacity for energy dissipation

⇒ Choice of reinforcing the structure – class 1 cross sections



1. Introduction
2. Design & Optimization
3. FEM
4. Ship impact
5. Ship impact results
6. Conclusion

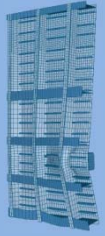
## 1st scenario: impact at U.W.L.

### Reinforced structure

- Dimensions of frames and girders increased
- Slenderness ratio **EC Class-1** cross section
  - Fully plastic bending moment
  - Large rotations
  - Take advantage of a yielding behaviour (dissipate large amount of energy)
- Structure total weight: 52 t → 69 t (+34%)  
Total cost +14%

Modèle de comportement	Mom de résistance	Capacité de rotation	Classe
	Moment plastique sur section brute 		1
	Moment plastique sur section brute 		2
	Moment élastique sur section brute 		3
	Moment plastique sur section efficace 		4

$M_{el}$  moment de résistance élastique de la section transversale  
 $M_{pl}$  moment de résistance plastique de la section transversale  
 $M$  moment appliqué  
 $\phi$  rotation (courbure) de la section  
 $\phi_{pl}$  rotation (courbure) de la section exigée pour générer une distribution plastique totale des contraintes dans la section transversale

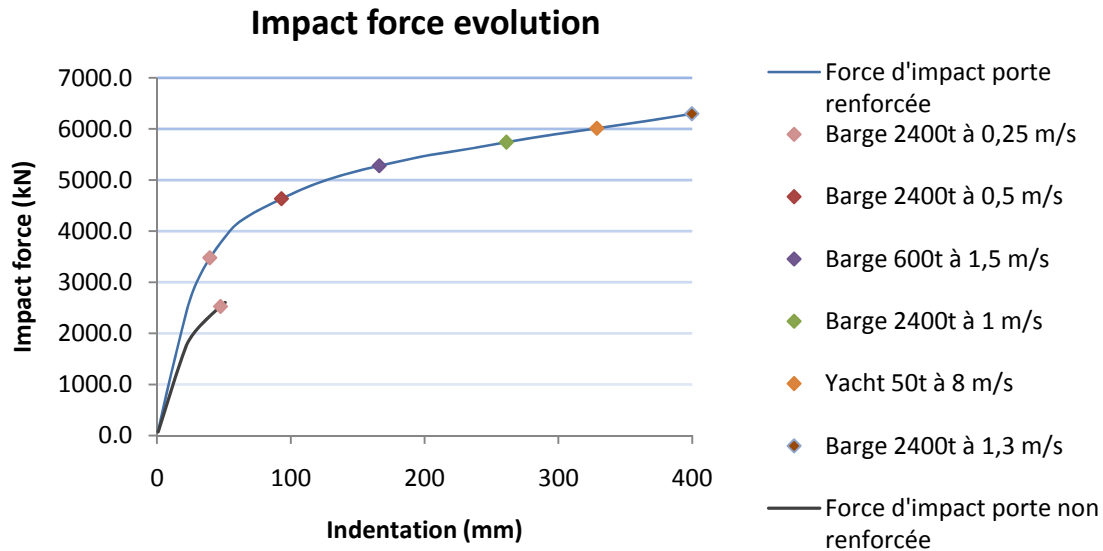


1. Introduction
2. Design & Optimization
3. FEM

4. Ship impact
5. Ship impact results
6. Conclusion

## 1st scenario: impact at U.W.L.

- With the reinforced structure (class 1)



- ⇒ Global plastic failure mechanism
- ⇒ Ductile global behavior
- ⇒ High capacity for energy dissipation

⇒ Reinforcing the structure with class 1 cross sections was a good choice



1. Introduction
2. Design & Optimization
3. FEM
4. **Ship impact**
5. Ship impact results
6. Conclusion

## Reinforced structure to avoid instability phenomenon – increase ductility

Ductile behaviour – very significant capacity for energy dissipation

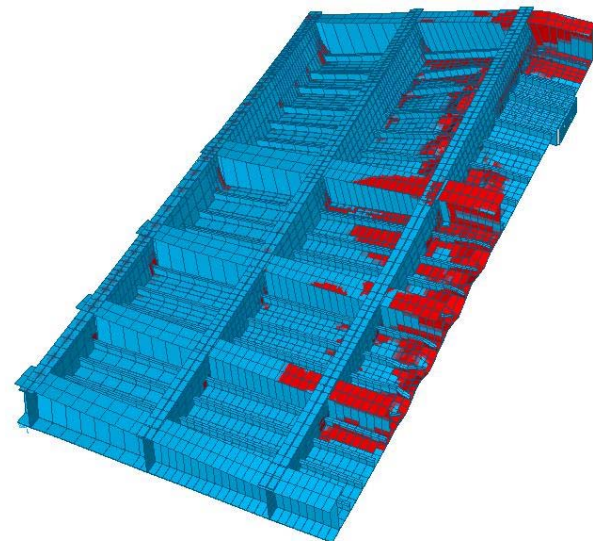
Initially optimized structure: 0,08 MJ

Reinforced structure: 2 MJ (i.e. a 2,400 t barge at 1.3 m/s)

### Yielding at the collapse stage



Initially designed structure



Reinforced structure



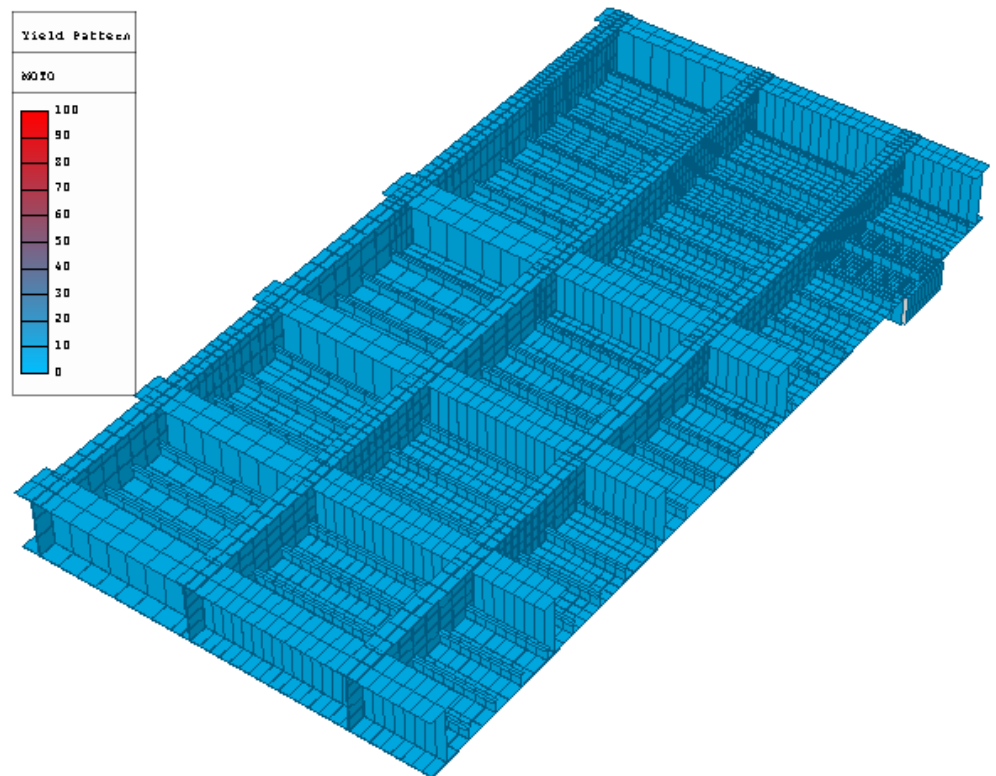
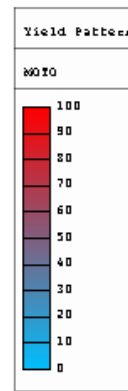
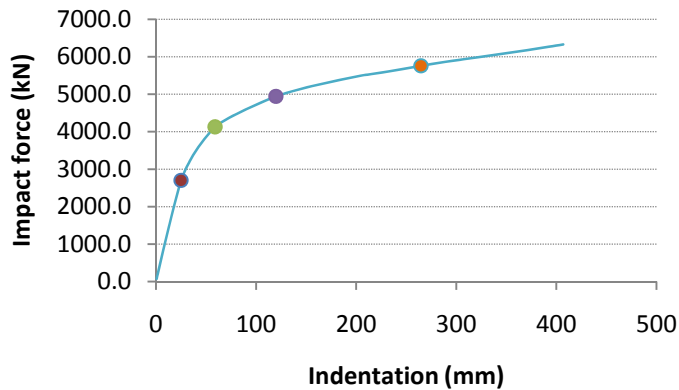


1. Introduction
2. Design & Optimization
3. FEM

4. Ship impact
5. Ship impact results
6. Conclusion

## Global plastic failure mechanism

- Highlighted by the strain state (yielding)
- Apparition of successive plastic hinges in the girders



→ The capacity for energy dissipation of the lock gate is strongly related to the ductility of its frames and girders

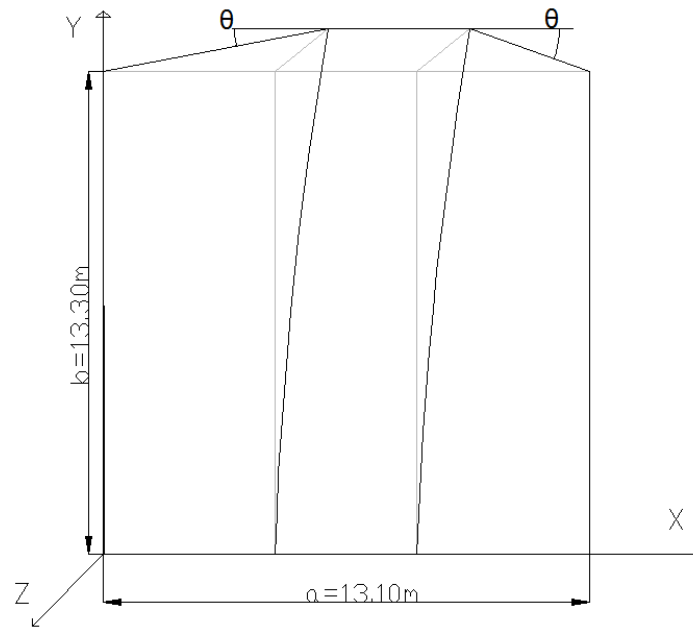


1. Introduction
2. Design & Optimization
3. FEM

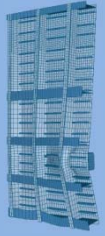
4. **Ship impact**
5. Ship impact results
6. Conclusion

## Global plastic failure mechanism

- 2 plastic hinges lines along the gate height
- This will allow us to calculate analytically an estimation of the gate strength



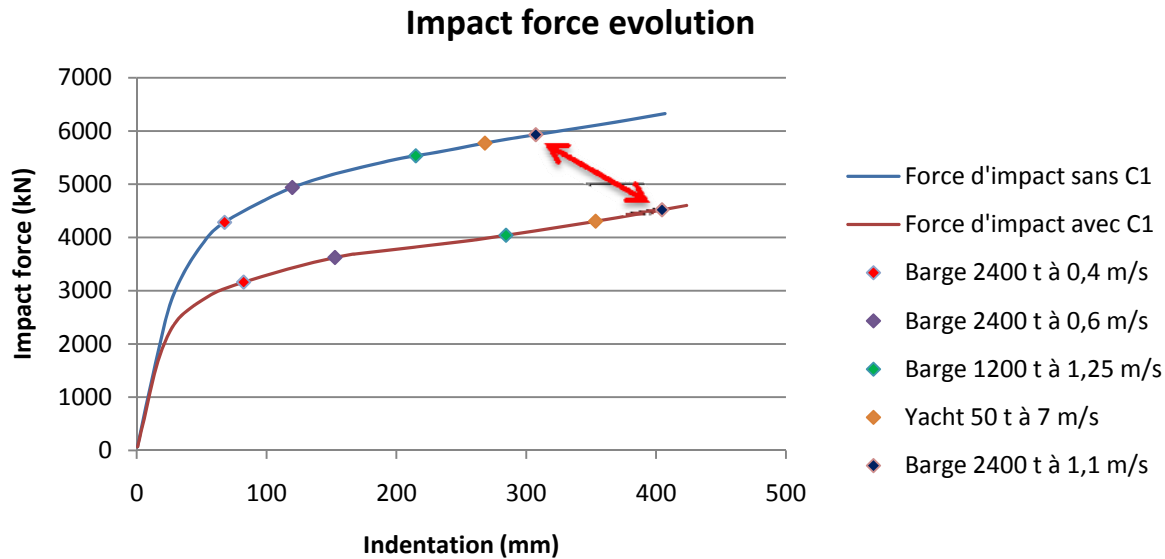
Simplified analytical representation of the global plastic failure mechanism



1. Introduction
2. Design & Optimization
3. FEM

4. Ship impact
5. Ship impact results
6. Conclusion

## 2<sup>nd</sup> scenario: impact at U.W.L. + hydrostatic load



The global behaviour of the gate is identical but the structure is previously submitted to a stress field

=> For a same impact (initial kinetic energy):

- Indentation ↗
- Impact force ↘
- Yielding in the structure ↗



1. Introduction
2. Design & Optimization
3. FEM

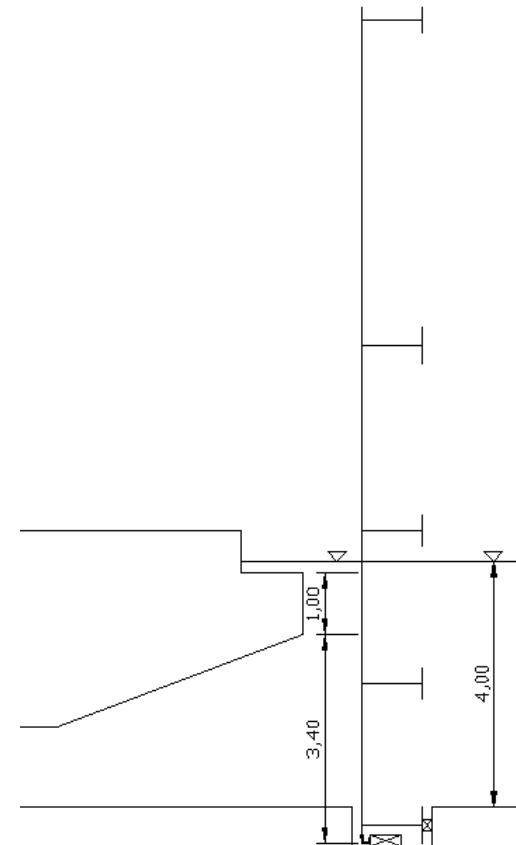
4. Ship impact
5. Ship impact results
6. Conclusion

## 3rd scenario: impact at D.W.L.

No additional hydrostatic load

Allows analyzing the influence of the impact zone

Highly stiffened impact zone



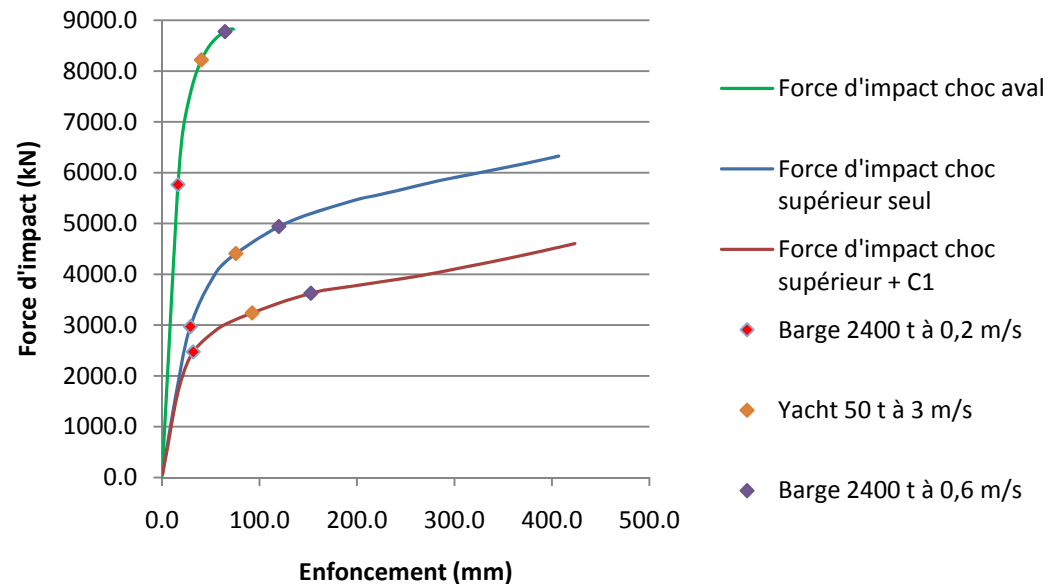


1. Introduction
2. Design & Optimization
3. FEM

4. Ship impact
5. Ship impact results
6. Conclusion

## 3rd scenario: impact at D.W.L.

- ✓ Fragile global behaviour
- ✓ For a same kinetic energy brought:
  - Impact force much higher
  - Deflection much lower
  - stiffer behaviour
- ✓ Lower capacity for energy dissipation







1. Introduction
2. Design & Optimization
3. FEM
4. Ship impact
5. Ship impact results
6. Conclusion

## 3rd scenario: impact at D.W.L.

Strain concentration in the impact zone leads to a fragile, sudden collapse

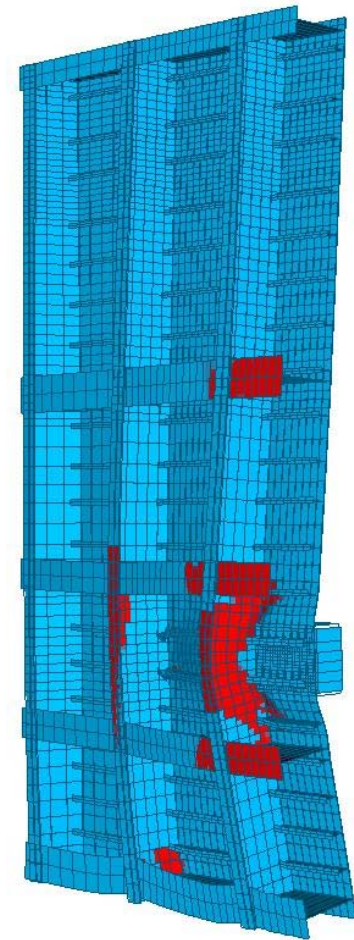
Modèle de comportement	Mom. de résistance	Capacité de rotation	Classe
	Moment plastique sur section brute 		1

Transverse stiffness  $\ll$  Longitudinal stiffness

⇒ No propagation of yielding

⇒ No global plastic failure mechanism

⇒ Collapse for a small indentation and low energy dissipation (0.5 MJ)





1. Introduction
2. Design & Optimization
3. FEM

4. Ship impact
5. Ship impact results
6. Conclusion

## Analytical assessment of the impact strength (1st scenario)

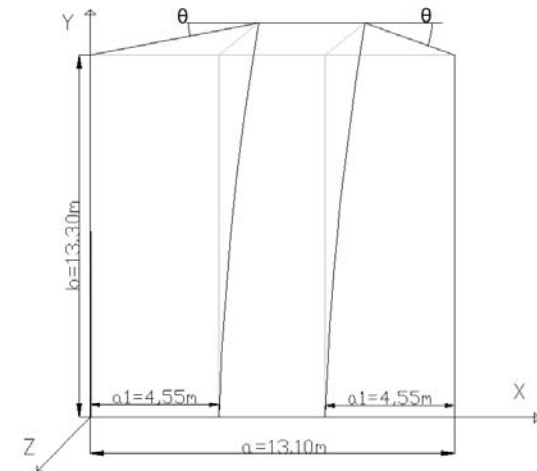
- ✓ Global plastic failure mechanism highlighted by the FEM analysis
- ✓ Analytical method developed by Le Sourne et al. gives an estimation of the impact strength
- ✓ Bending energy rate along the two plastic hinge lines:

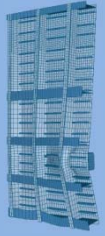
$$\dot{E}_b = 2 \int_0^l \tilde{M}_0 \times \dot{\theta} dl$$

$$\dot{E}_b = \frac{2}{\alpha_1} \int_0^l \tilde{M}_0 \times w dl$$

- ✓ 2 unknowns:

- The fully plastic bending moment per unit length of the gate
- The displacement field





1. Introduction
2. Design & Optimization
3. FEM

4. Ship impact
5. Ship impact results
6. Conclusion

## Analytical assessment of the impact strength (1st scenario)

### Assumptions:

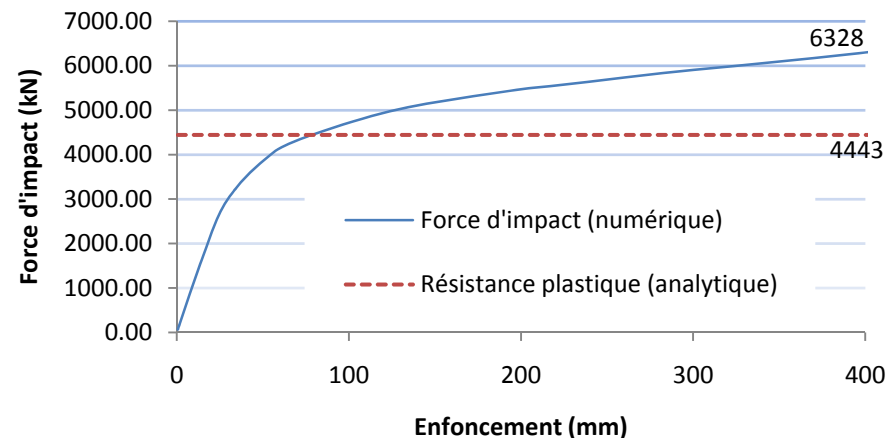
- Neglect the variation of the fully plastic moment per unit length along the plastic hinges
- Assume a linear displacement field

It holds:

$$P_{gb} = \frac{M_p}{a_1} \frac{b}{h_{impact}} = 4\,443 \text{ kN}$$

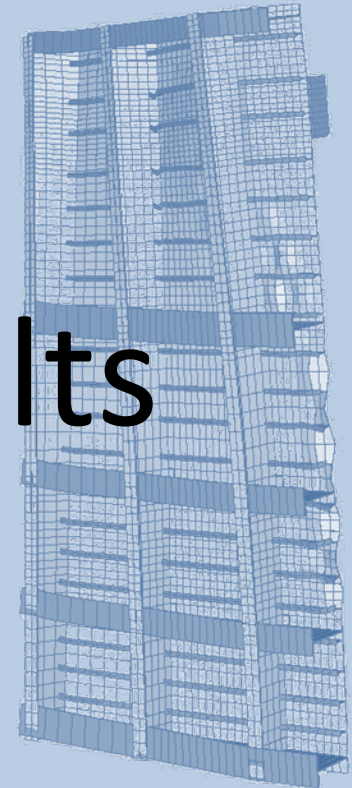
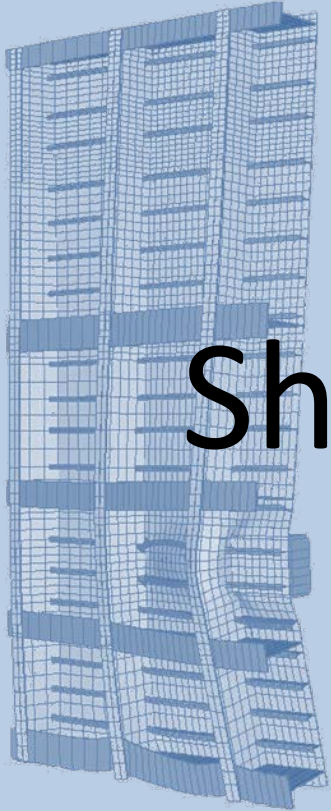
Numerical analysis strength overtakes by 42 % analytical strength

New mode of strength (membrane)?  
Effect of the assumptions and simplifications?





# Ship impact results





1. Introduction
2. Design & Optimization
3. FEM
4. Ship impact
- 5. Ship impact results**
6. Conclusion

## Presentation of the results

1. Effect of a determined impact on the gate for the different studied cases

Impact of a 1,200 t barge at 0,8 m/s (384 kJ)	U.W.L. without hydrostatic load	U.W.L. with hydrostatic load	D.W.L.
Impact force	4,845 kN	3,550 kN	8,706 kN
Indentation (only due to the impact)	11.1 cm	13.9 cm	5.9 cm
Number of plastic hinges in frames and girders	2 girders	3 girders	1 frame





1. Introduction
2. Design & Optimization
3. FEM
4. Ship impact
5. **Ship impact results**
6. Conclusion

## Presentation of the results

2. Maximum impact speed of a ship of determined weight for the different studied cases

Impact speed (m/s)	U.W.L. without hydr. load (2074 kJ)	U.W.L. with hydr. load (1538 kJ)	D.W.L. (495 kJ)
Yacht 50 t	9.11	7.84	4.45
Barge 600 t	2.63	2.26	1.28
Barge 1,200 t	1.86	1.60	0.91
Barge 2,400 t	1.31	1.13	0.64

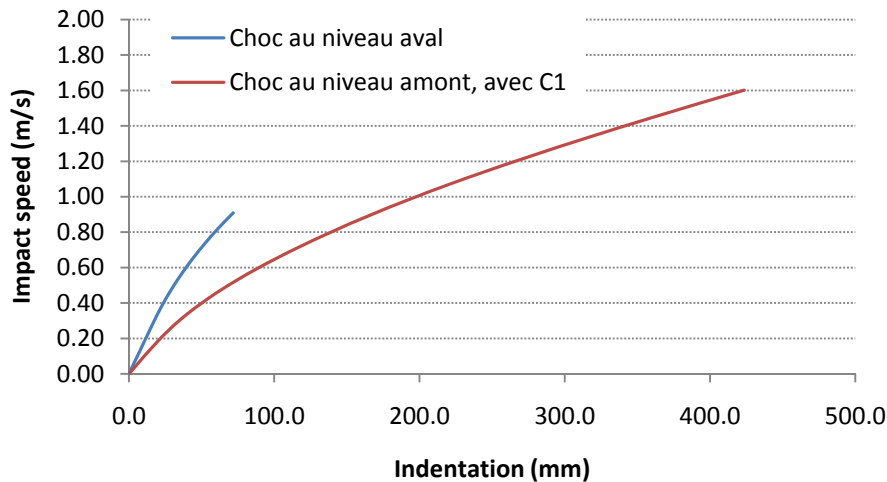


1. Introduction
2. Design & Optimization
3. FEM
4. Ship impact
5. **Ship impact results**
6. Conclusion

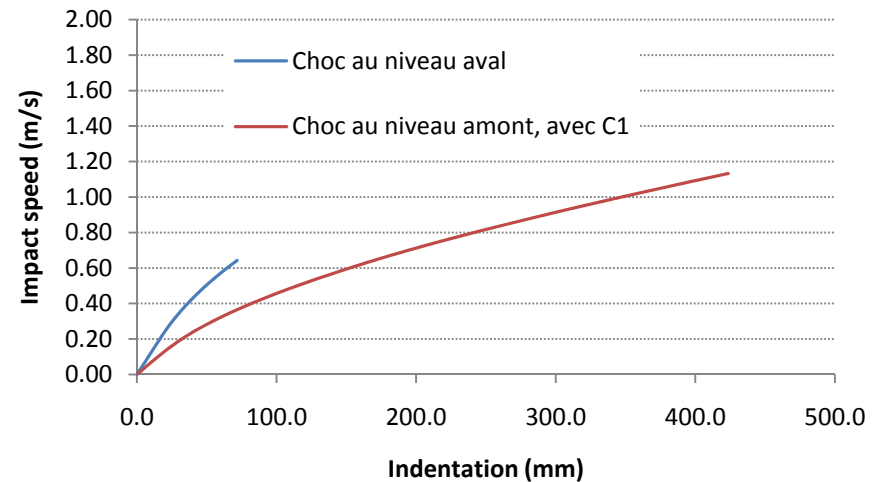
## Presentation of the results

3. Relationship between the impact speed and the indentation of a ship of determined weight for the different studied cases

Relationship impact speed – indentation (m=1,200t)



Relat. impact speed – indentation (m=2,400t)



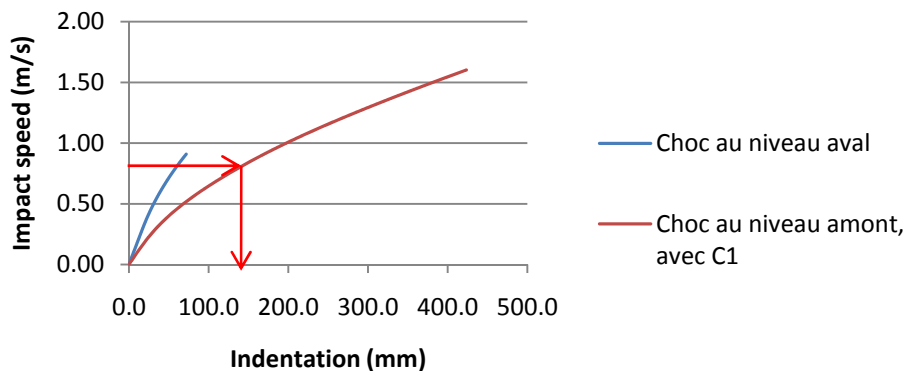


1. Introduction
2. Design & Optimization
3. FEM
4. Ship impact
5. **Ship impact results**
6. Conclusion

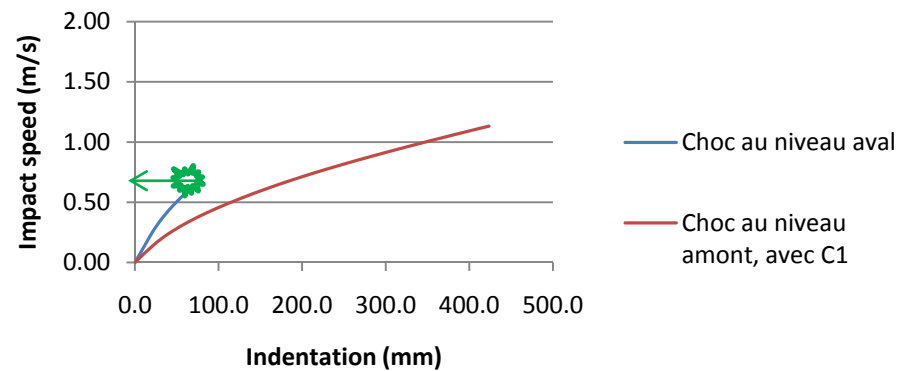
Impact of a 1,200 t barge at 0,8 m/s (384 kJ)	U.W.L. without hydrostatic load	U.W.L. with hydrostatic load	D.W.L.
Impact force	4,845 kN	3,550 kN	8,706 kN
Indentation (only due to the impact)	11.1 cm	13.9 cm	5.9 cm
Number of plastic hinges in frames and girders	2 girders	3 girders	1 frame

Impact speed (m/s)	U.W.L. without hydr. load (2074 kJ)	U.W.L. with hydr. load (1538 kJ)	D.W.L. (495 kJ)
Yacht 50 t	9.11	7.84	4.45
Barge 600 t	2.63	2.26	1.28
Barge 1,200 t	1.86	1.60	0.91
Barge 2,400 t	1.31	1.13	0.64

Relationship impact speed - indentation (m=1,200t)



Relat. Impact speed - indentation (m=2,400t)

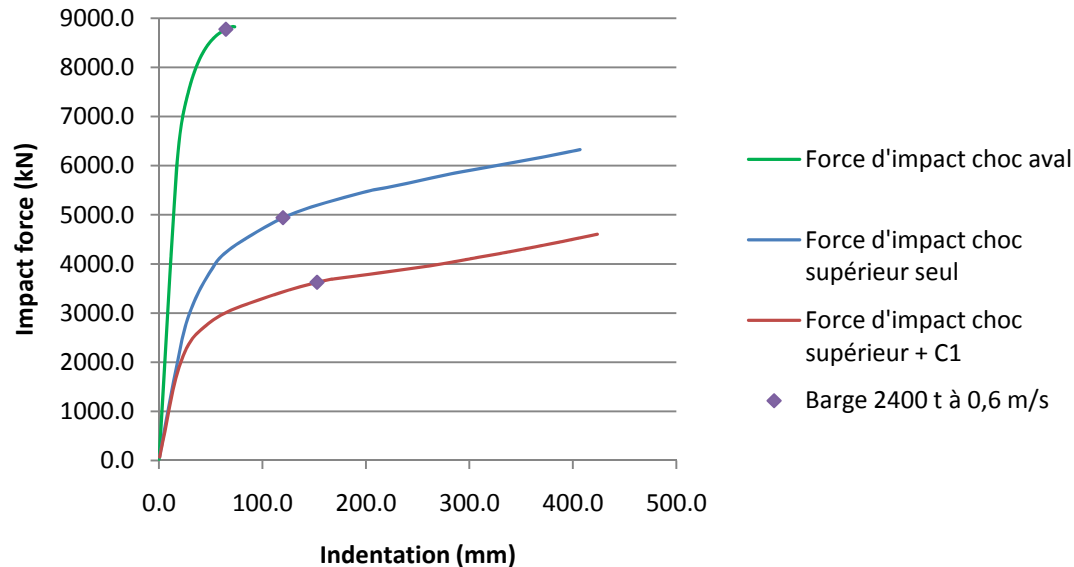


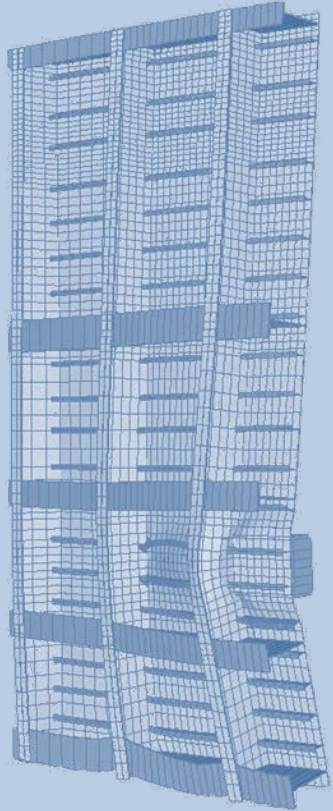


1. Introduction
2. Design & Optimization
3. FEM
4. Ship impact
5. **Ship impact results**
6. Conclusion

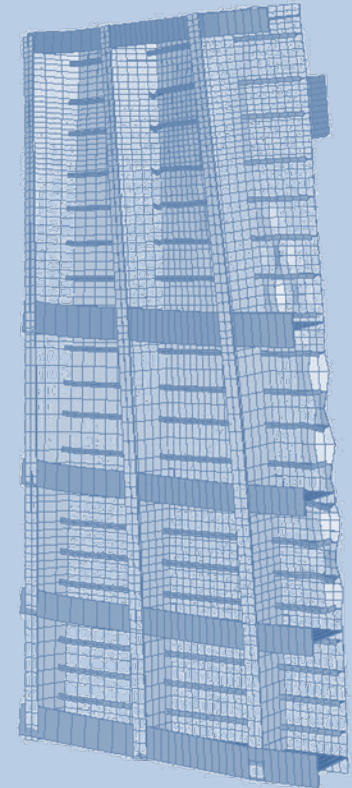
## What did we learn from the analysis of the ship impact?

- ✓ To dissipate energy, it needs ductility => interest of the global plastic failure mechanism
  - Ductility of the elements can be achieved by using class-1 cross sections
  - Ductility of the gate requires a good propagation of yielding, which can be achieved by a good design of the stiffness ratios in the potential impact zones





# Conclusion

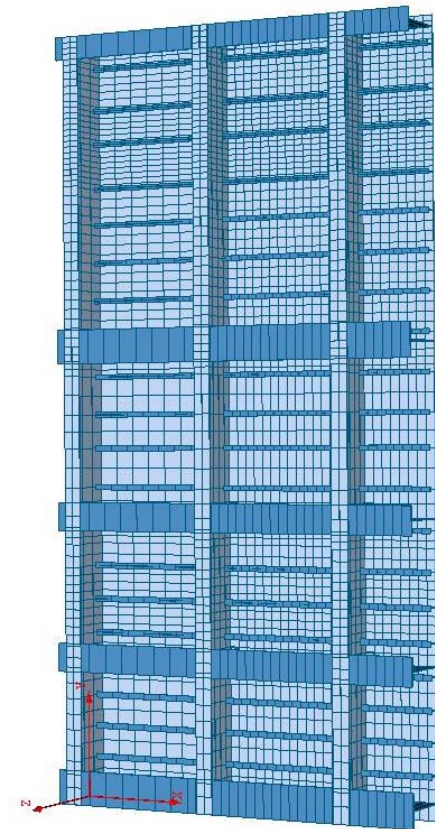
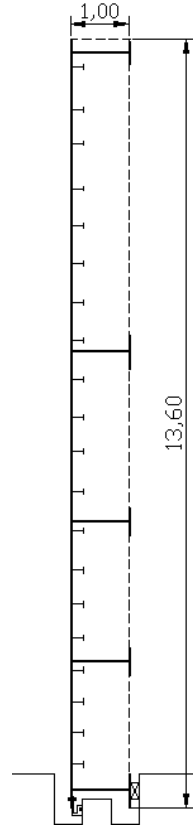
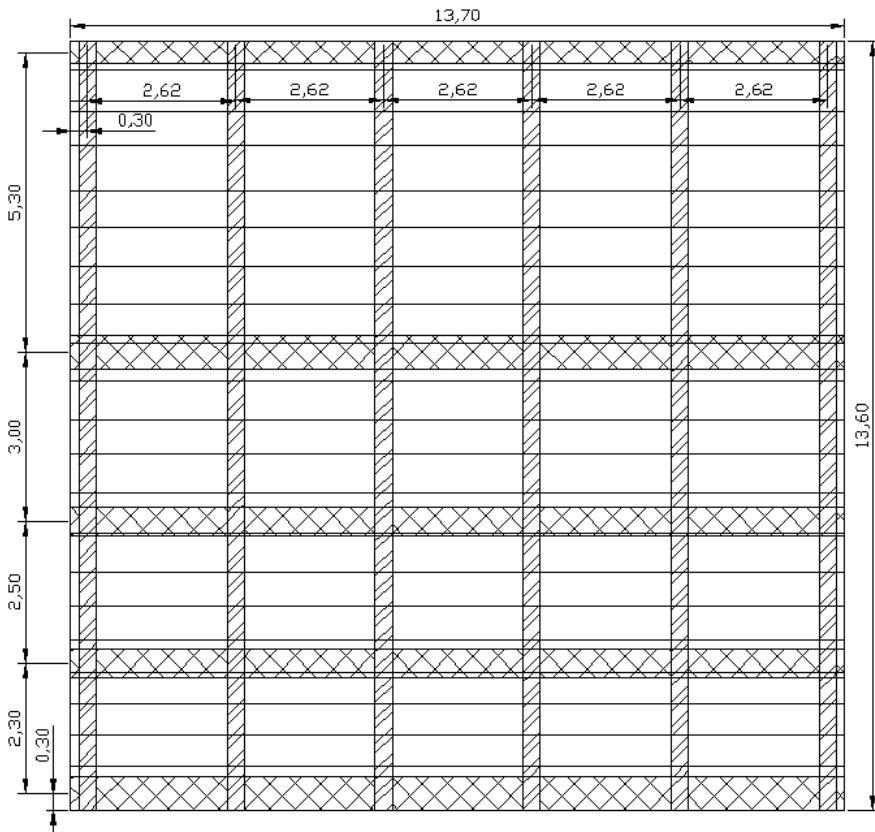






1. Introduction
2. Design & Optimization
3. FEM
4. Ship impact
5. Ship impact results
6. Conclusion

## Design and Optimization





1. Introduction
2. Design & Optimization
3. FEM
4. Ship impact
5. Ship impact results
6. **Conclusion**

**1° objective : design and optimization of the downstream lock gate**

⇒ **Elastic design considering hydrostatic load case**

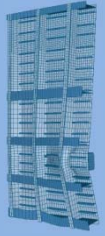
**2° objective : ship impact analysis (gate = ship stopping device)**

⇒ **Analysis of 3 scenario highlighted the following:**

- **Interest of class-1 cross sections**
- **Influence of the impact zone (stiffness)**
- **Influence of the hydrostatic load (previous stress state)**

⇒ **Increase dimensions of frames and girders (weight +34%, cost +14%)**

⇒ **2 kinds of global behavior: Fragile VS Ductile**



1. Introduction
2. Design & Optimization
3. FEM
4. Ship impact
5. Ship impact results
6. Conclusion

## Analysis of the final (reinforced) structure for hydrostatic loads

Loading	Service	Flood	Downstr. empty
Girders (MPa)	122	137	129
Stiffeners (MPa)	156	168	171
Frames (MPa)	107 – <b>112</b>	114 – <b>118</b>	<b>120 – 126</b>
Plate (ép./ép. min)	1,07	1,02	1,05
Maximum deflection	1,50 cm – <b>1,79 cm</b>	1,69 cm – <b>2,00 cm</b>	1,58 cm – <b>1,90 cm</b>

LBR5 analysis of the reinforced structure (**bold = FINELG**)

- ✓ Stiffeners and plate still optimum ( $\sigma \approx 175$  MPa)
- ✓ Frames and girders are not optimized because reinforced (class-1) for impact
  - ⇒ Iterative process: ship impact analysis – optimization
  - ⇒ Or introduce new constraints in the optimization software to obtain optimized solutions considering impact strength



1. Introduction
2. Design & Optimization
3. FEM
4. Ship impact
5. Ship impact results
6. **Conclusion**

## Ship impact: new constraints for optimization process?

**Aim : dissipate energy => ductility**

**U.W.L. impact on the initially designed structure (classe 3)**

**U.W.L. impact on reinforced structure (classe 1)**

**U.W.L. impact on reinforced structure (classe 1), with hydrostatic load**

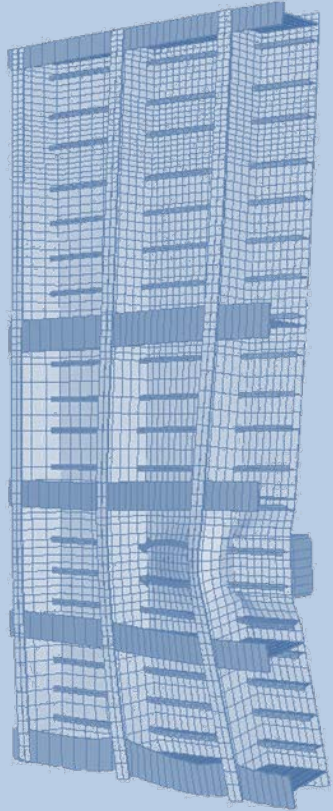
**D.W.L. impact on the reinforced structure (classe 1)**

**⇒ Use class-1 cross sections**

**⇒ Good design of the stiffness ratios in the impact zones**

**⇒ ... ? Maybe other criteria?**

**⇒ Finally, develop an optimization software integrating ship impact to be able to compare directly the optimized reinforced solution VS the optimized elastic solution coupled with protective measures against ship impact**



Thank you

